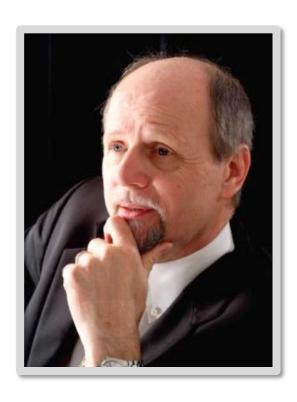
Designing Transducers for Compact Active Speakers



MEET THE EXPERTS - LEARN THEIR SECRETS



Peter Larsen with these companies from 1974



Peter Larsen LOUDSOFT















The Design Cycle



Designing Motors for Small Drivers



Bass Alignments in Box @ High Power



Cone Designs and Problems



Crossover Designs in Practice



Loudspeaker Measurement Examples



Challenges in Speaker QC Testing



LOUDSOFT

Designing Transducers for Compact Speakers





BBC LS 3/5A Compact Monitor 1974





Designing Transducers for Compact Speakers





Todays Compact Loudspeakers





Table 1. SUMMARY OF LOUDSPEAKER ALIGNMENTS adapted from Table 1, Loudspeakers in Vented Boxes

No.	Type	Ripple (dB)	f ₃ /f _s	f _B /f _S	C _{AS} /C _{AB}	Q_T	faux/fs
1	QB3		2.68	2.00	10.48	0.180	
2	QB3		2.28	1.73	7.48	0.209	
3	QB3		1.77	1.42	4.46	0.259	
4	QB3	-	1.45	1.23	2.95	0.303	
5	BW4	- I	1.000	1.000	1.414	0.383	-
6	CH4		0.852	0.927	1.055	0.415	9
7	CH4	0.07	0.724	0.829	0.729	0.466	-
8	CH4	0.25	0.704	0.757	0.559	0.518	
9	CH4	0.51	0.685	0.716	0.485	0.557	- 1
10	BW5	1 2	1.000	1.000	1.000	0.447	1.00
11	CH5		0.850	0.912	0.583	0.545	1.22
12	CH5	0.25	0.698	0.814	0.273	0.810	1.81
13	CH5	0.5	0.620	0.798	0.227	0.924	200
14	CH5	1.0	0.554	0.781	0.191	1.102	

QB3 ≡ Quasi-Butterworth 3rd order

BW4 ≡ Butterworth 4th order, maximally-flat amplitude response

CH4 ≡ Chebyshev 4th order, equal-ripple amplitude response

BW5 ≡ Butterworth 5th order, maximally-flat amplitude response

CH5 ≡ Chebyshev 5th order, equal-ripple amplitude response

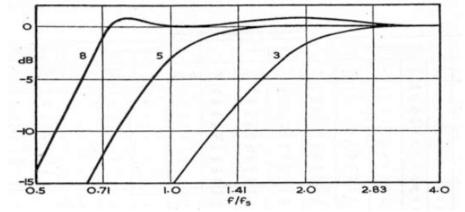


Fig 8 Amplitude Responses of a Vented Box Loudspeaker, with alignments numbered as in Table 1

From Neville Thiele: "The Loudspeaker Parameters and their Evolution"

Compact speakers require small drivers. These produce less SPL (lower sensitivity)



Initial settings for 6inch example A

1.Cone Area: 136cm2

2.Cone Mass (incl. ½ surround): 11g

3.DCR: 3.0 R (Nominal impedance: 4 ohms)

Initial settings for 3inch example B

1.Cone Area: 31.2cm2

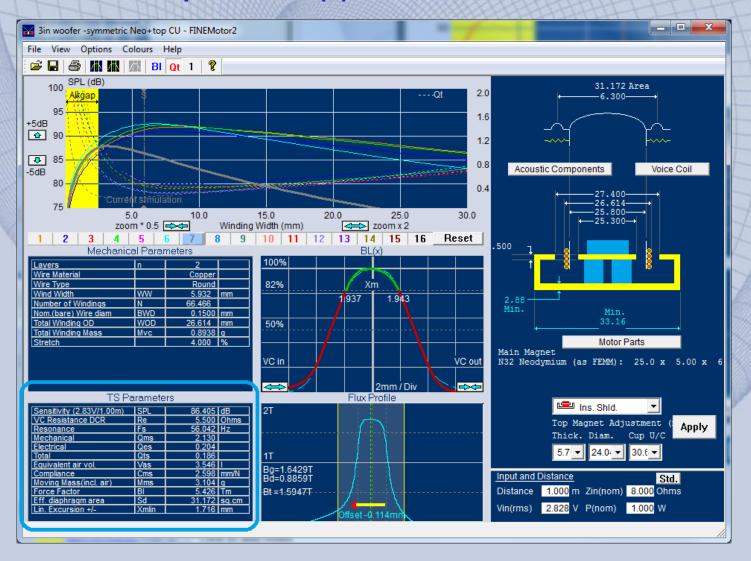
2.Cone Mass (incl. ½ surround): 2.2g

3.DCR: 5.5 R (Nominal impedance: 8 ohms)

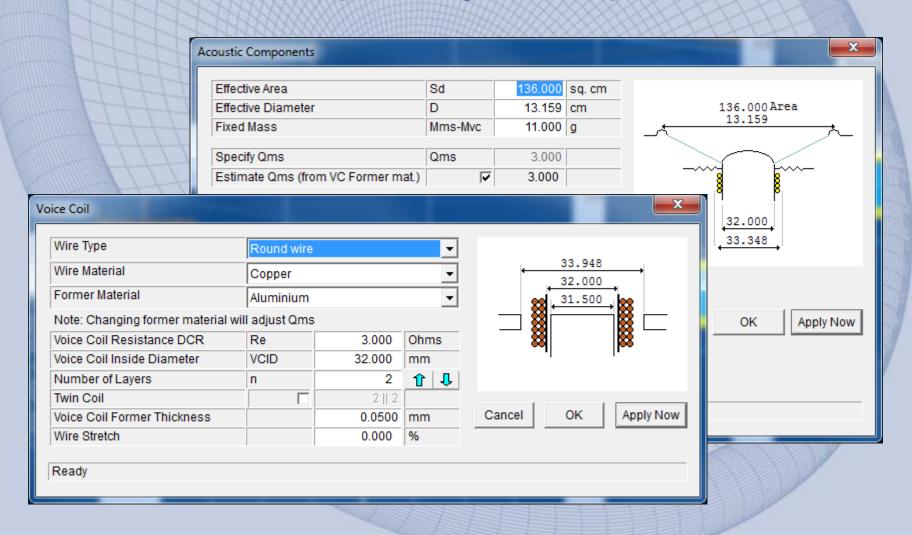
Example A:



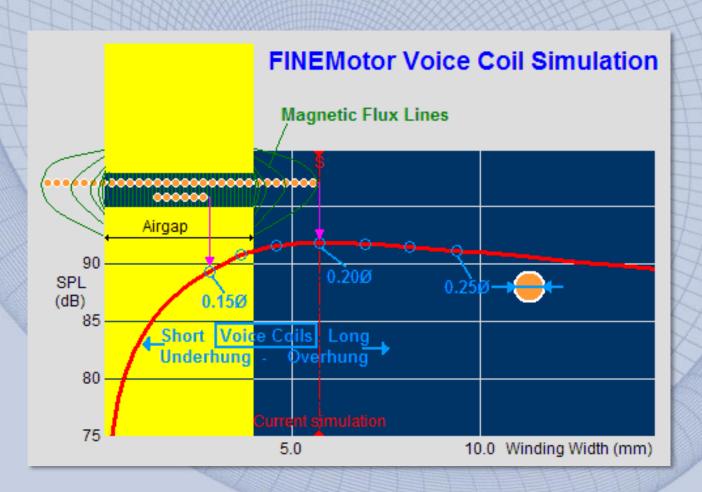
- 1.Target Xmax ~5mm
- 2. Optimize TS Parameters (Qts < 0.4)
- 3. Optimize BL(x) for Low Distortion



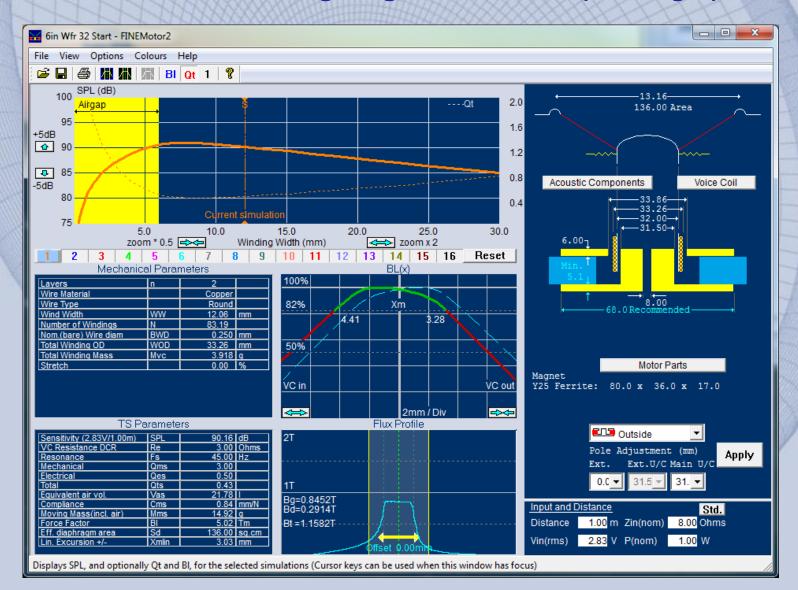
Initial input settings for example A



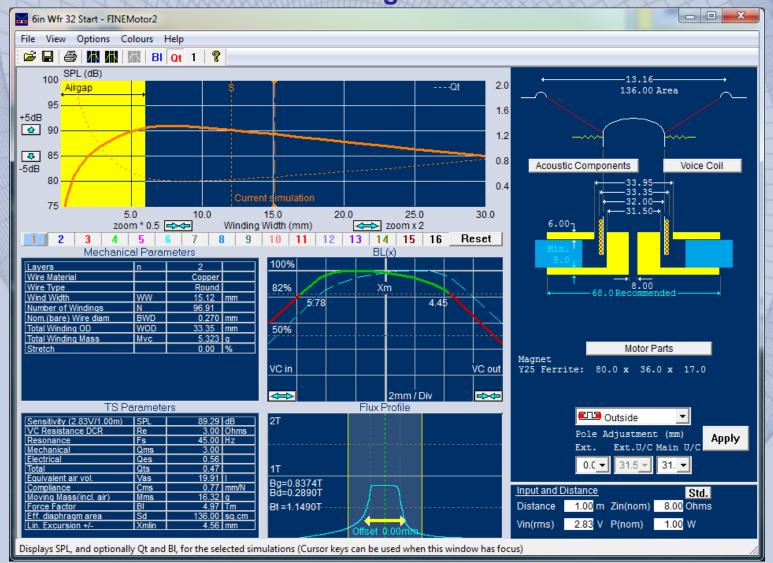
Calculated solutions based on Wire Diameters



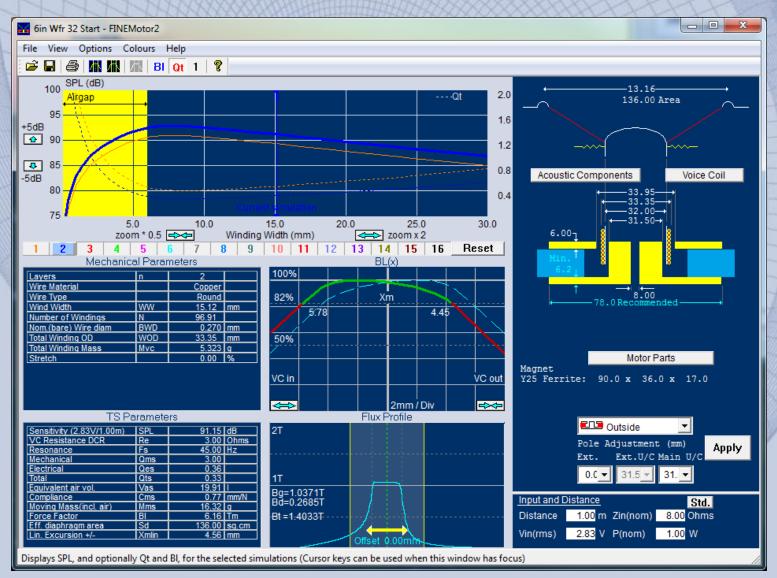
(A) Initial Solution: Xm is only 3mm. Move cursor to the right to find solutions having longer Voice Coils (Xm larger).



Now Xm is close to 5mm (OK). However Qts=0.47 is too high

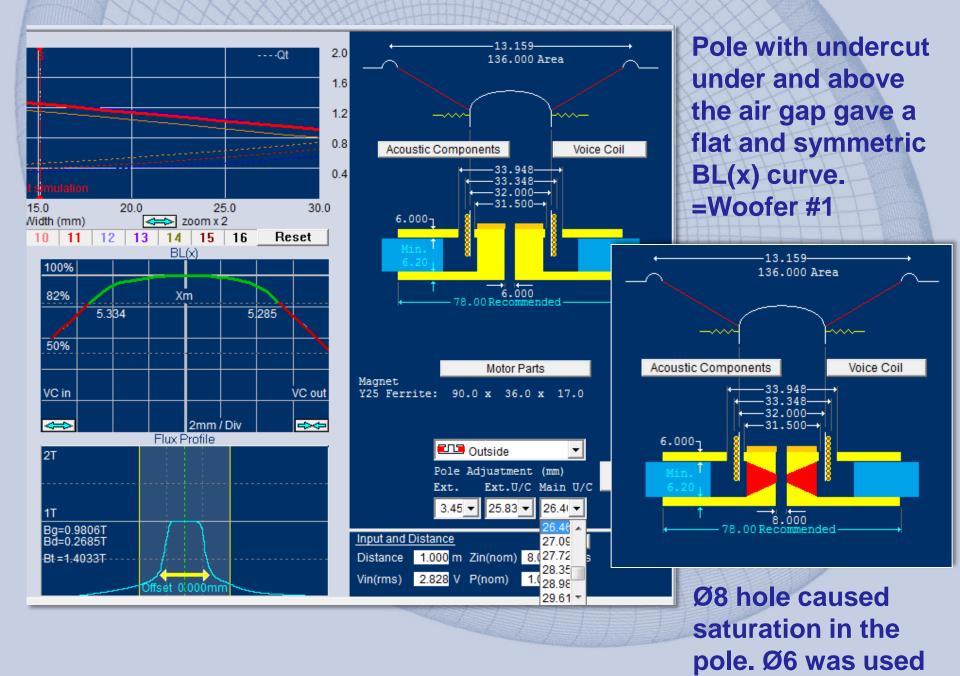


A larger 90x36x17 Y25 magnet gave Qts=0.33 © But the BL(x) curve is NOT symmetrical or flat

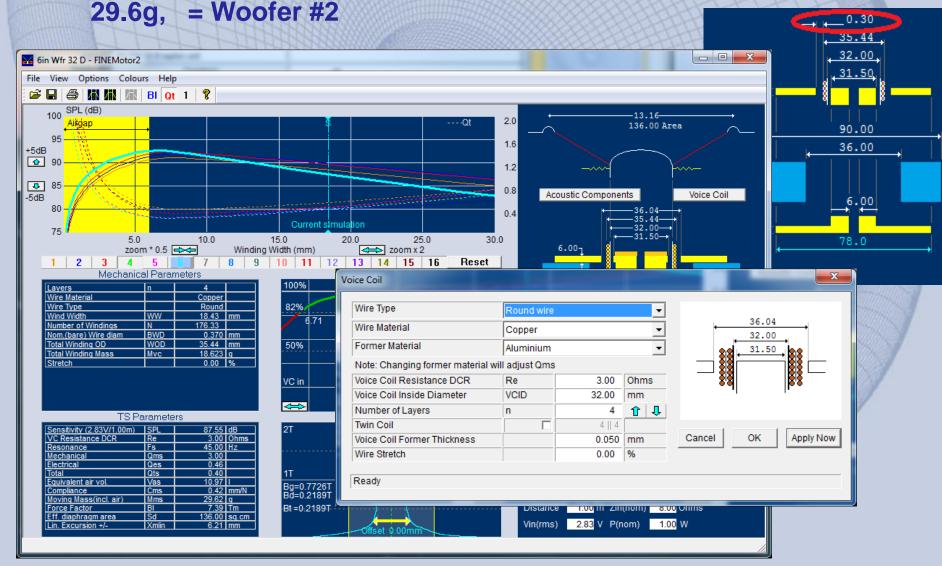


Extending the pole by 3.68mm produces a nice symmetric BL(x) curve, though not flat. But (IM) distortion will be reduced.

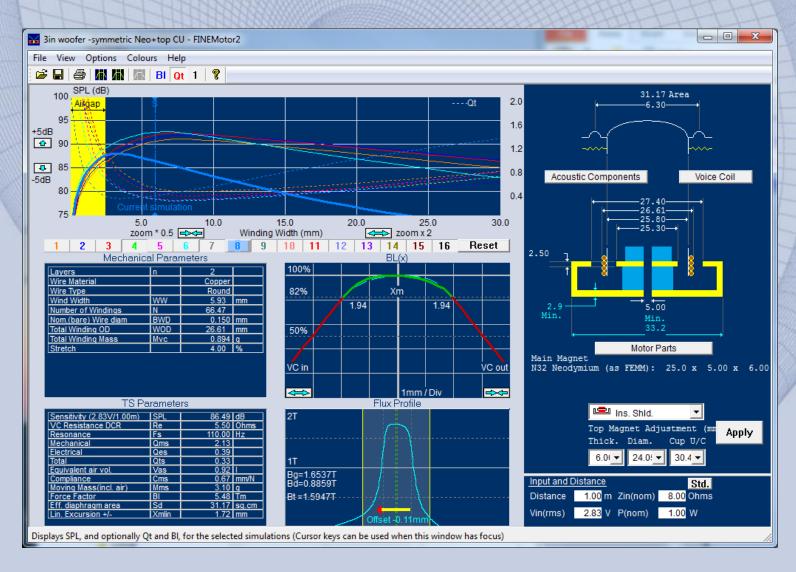




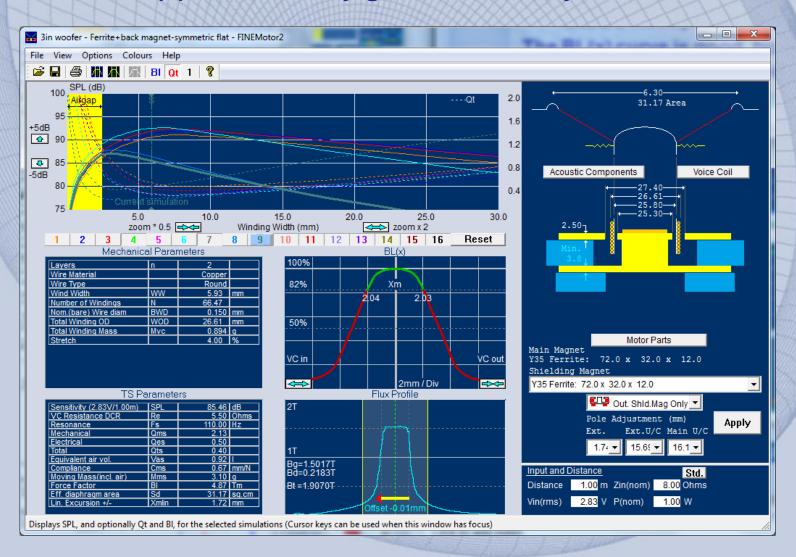
An alternative design using a 4-layer Voice Coil. The air gap OD was auto-adjusted for the larger Voice Coil using a 0.30mm clearance. Xm=+/-6.7mm. Moving Mass Mms up from 16.3 to



Example B. This small 3inch woofer is designed like before. But this time a Neodymium Magnet + a top Neo magnet was used. The BL(x) curve is good, but sensitivity is low. = Woofer #3



Alternative 3" Ferrite magnet + back magnet solution for low cost. The BL(x) curve is very good. Sensitivity is still low.



Question:

Which parameter influences Qts most?

Mms (Moving mass, Re (DCR) or BL (Force factor)







Feedback from You: Qts

Qts is defined as the total Q (quality factor) at the resonance Fs.

1/Qts= 1/Qms+1/Qes

Qes < Qms (typically 0.4 < 5)

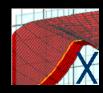
Qes=2pi*Fs*Re*Mms / BL^2

Answer is BL





Bass Alignments in Box @ High Power



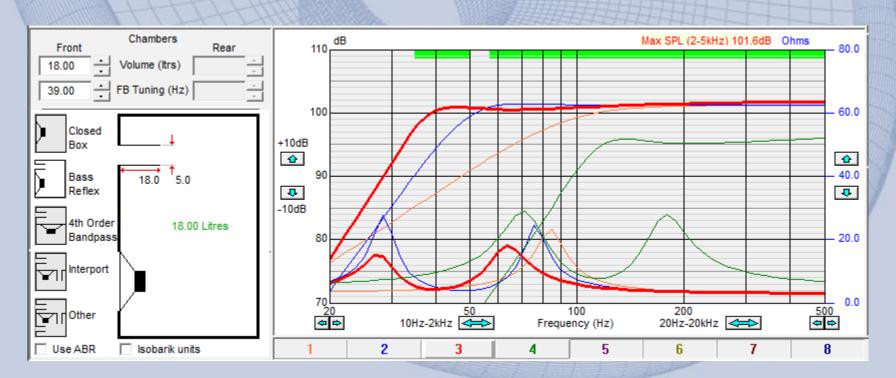
Next we will study how the designed woofers will behave in cabinets of different sizes and tuning. The effect of high input power will be included.



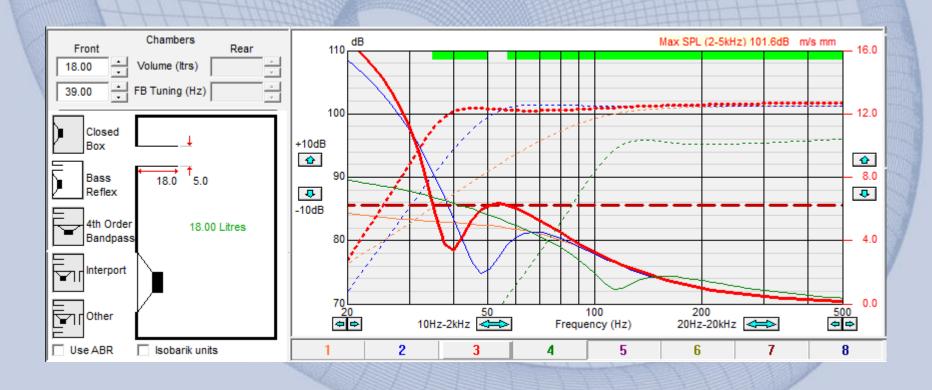


The 3 example drivers were imported into the FINEBox program:

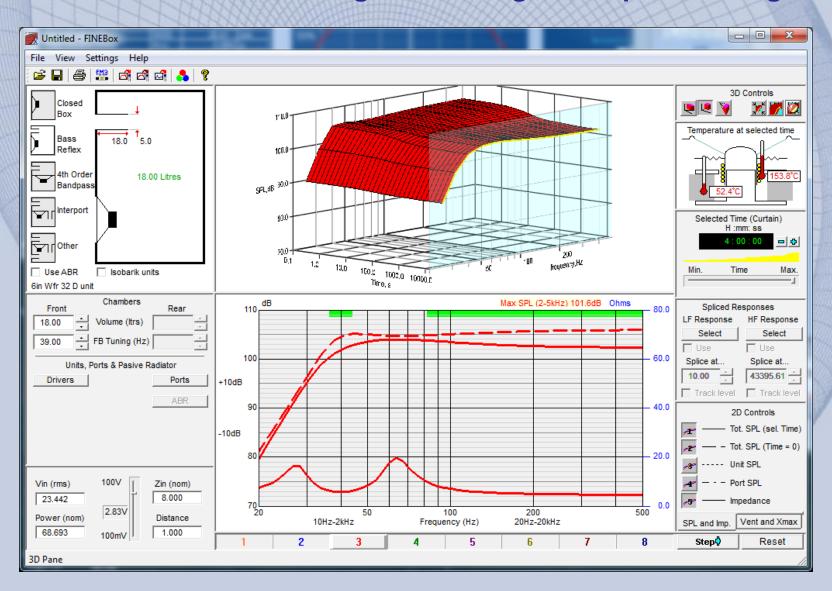
- Woofer #1 in 8 Ltr. closed box____
- 2. Woofer #1 in 18 ltr. Bass Reflex, tuned to Fb= 48Hz_
- 3. Woofer #2 in 18 ltr. Bass Reflex, tuned to Fb= 39Hz
- 4. Woofer #3 in 1 ltr. Bass Reflex, tuned to Fb= 115Hz_
- #3 has more bass extension, due to more mass and BL.



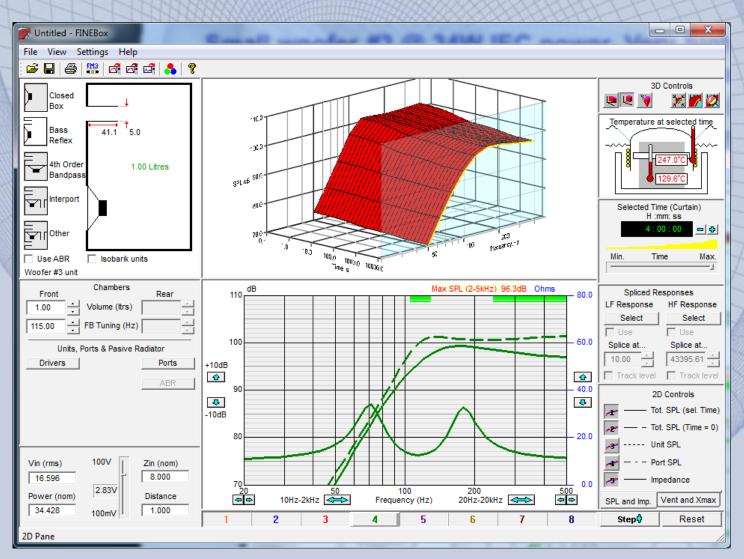
Bass Reflex Woofer Displacement reaching Xmax of woofer #2. LF rise to be filtered.



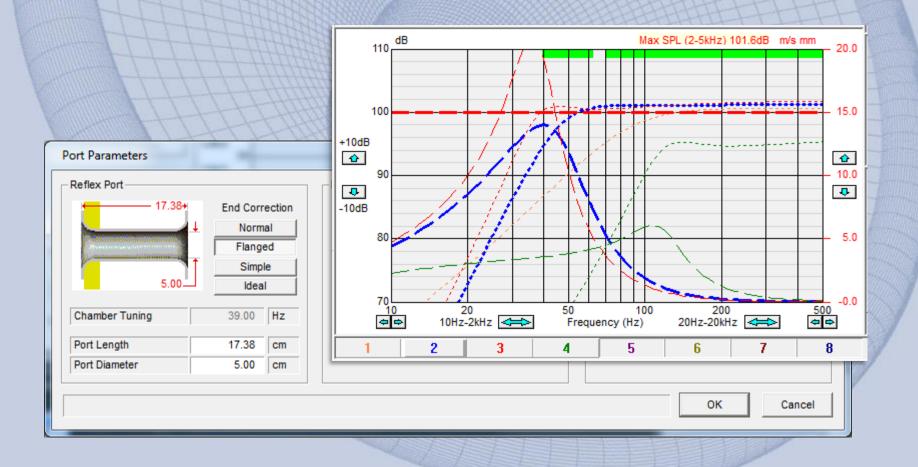
Power Compression of Woofer #2 @ 70W IEC input. The Voice Coil has reached 154 deg. C. The magnet temp is 52.4 deg. C

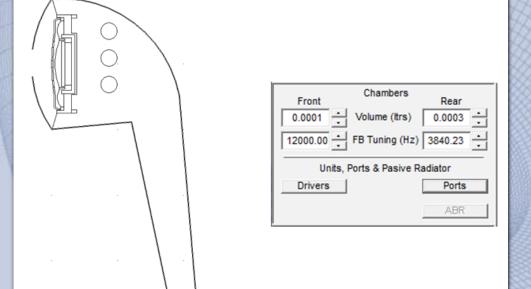


Small woofer #3 @ 34W IEC power. Very high Power Compression, and the Voice Coil is 247 deg. C = Overload! The neodymium magnet is probably demagnetized @ 129 deg. C



Bass Reflex Port dimensions to avoid air noise due to turbulence for curve No. 2 (Woofer #1)

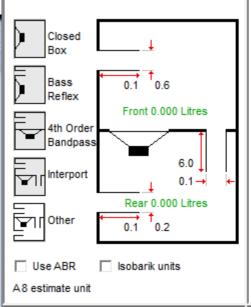


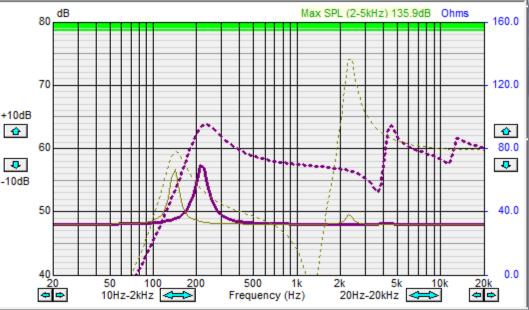


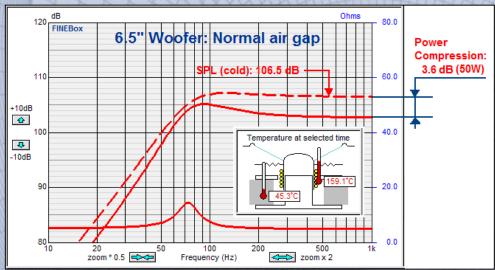
HEADPHONE DESIGN

Simplified lumped element simulation of Headphone /Earphone with cavities and holes/channels.

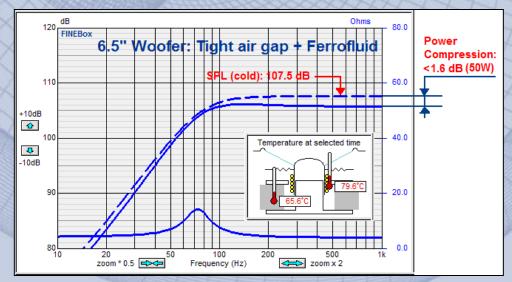
(Infinite baffle: No Coupler or Artificial Ear).





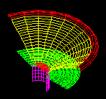


Calculated Compression of 61/2" Woofer



Reduced Compression with Ferrofluid + Tighter Air gap

Cone Designs and Problems



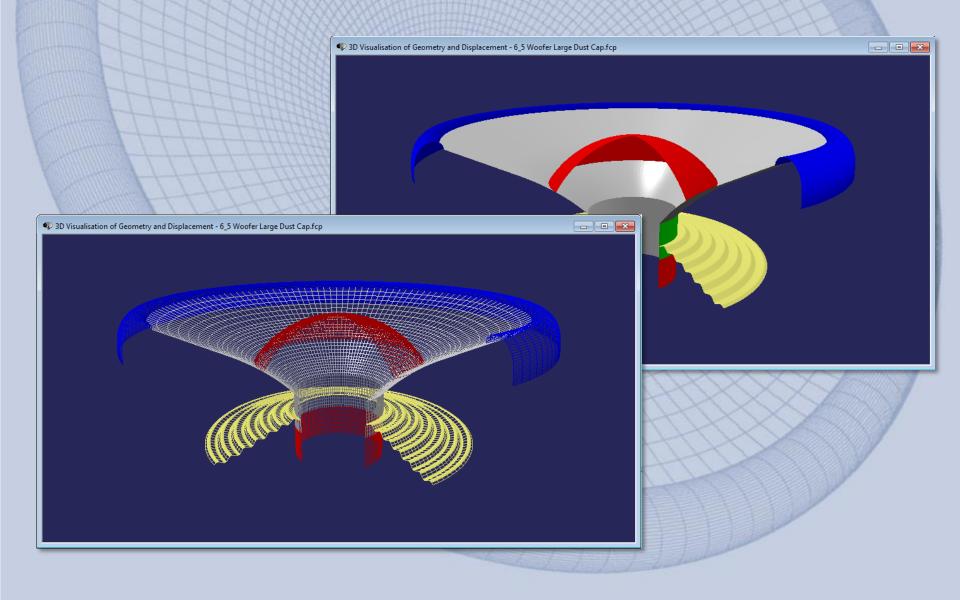
Cone design used to be a challenge based on trial and error.

Today we can quickly simulate with the help of FEM and gain insight into the mechanical and acoustical behavoir of cones and domes of any size

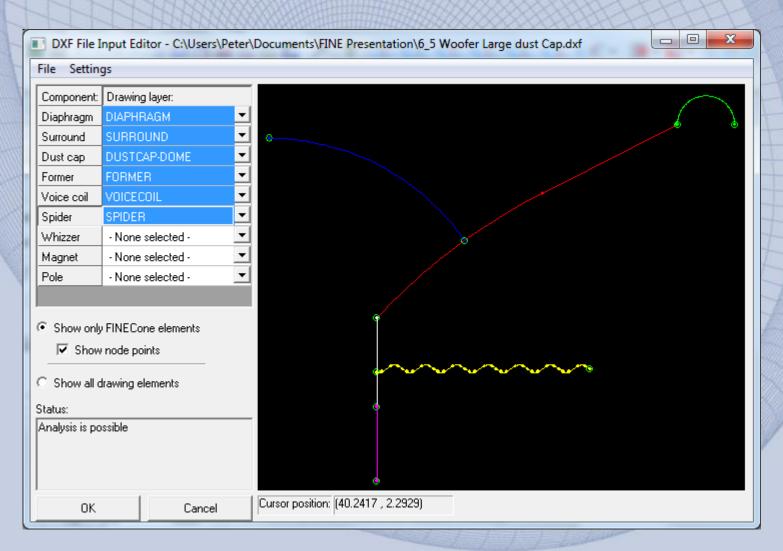




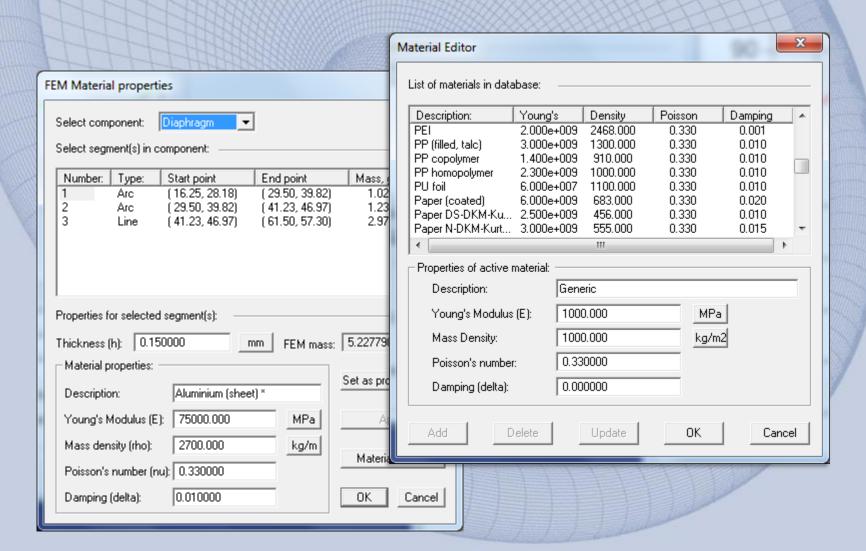
Acoustic Finite-Element (FEM) Simulation examples



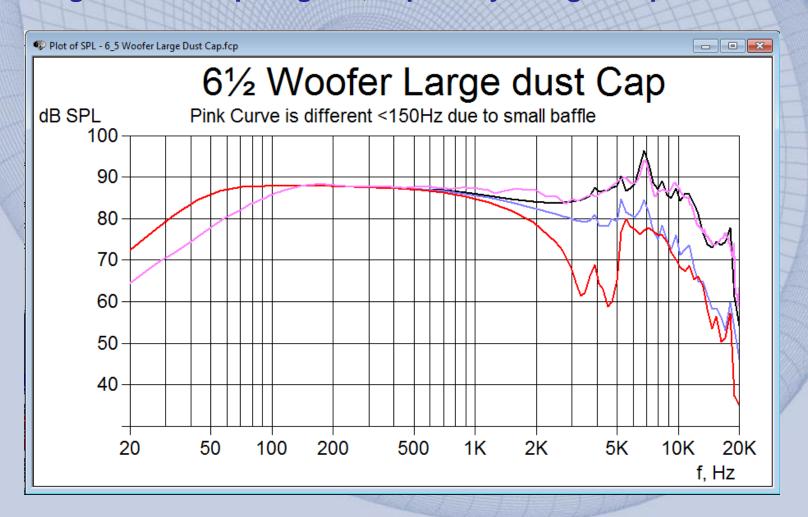
The driver Geometry can be defined as a simple DXF drawing



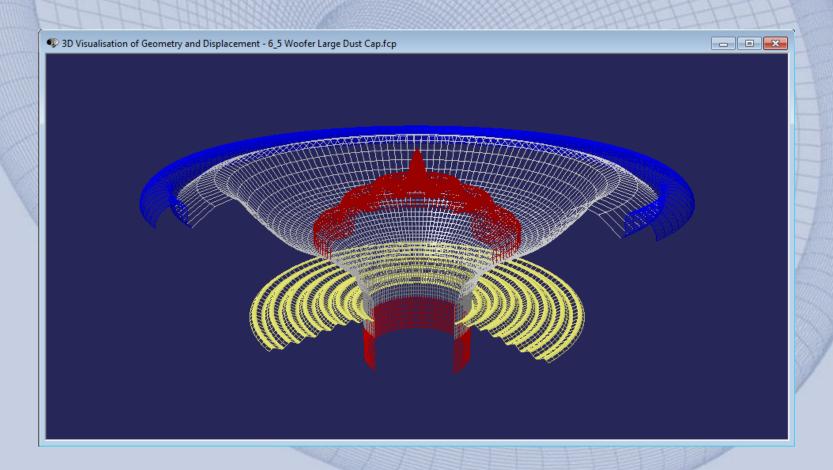
Each segment is given material parameters from a comprehensive database



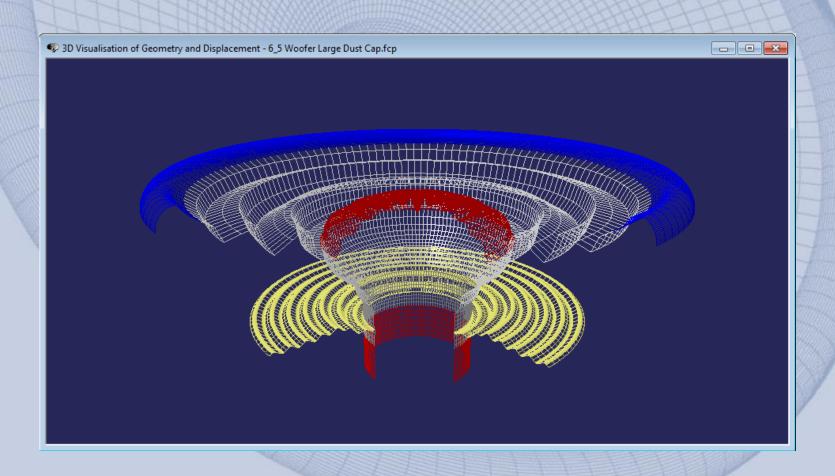
FEM simulated 0__/30__/60__ deg. frequency responses of 6½" Woofer, compared to actual measurement__. The agreement is quite good, especially at high frequencies.



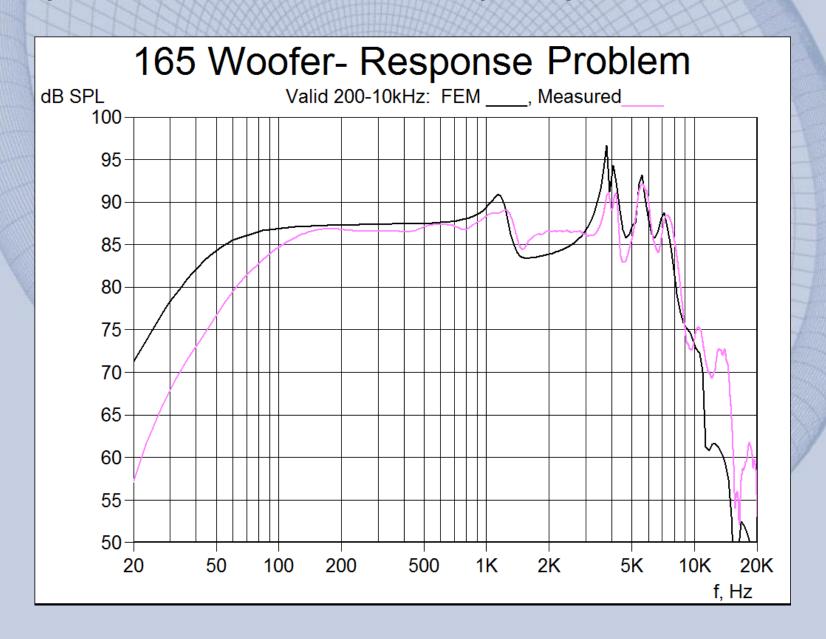
Cone and Dust cap Break-up of 6½" woofer @ 6804 Hz. The outer half of the cone shows the 1st cone mode, and the dust cap has high order break-up



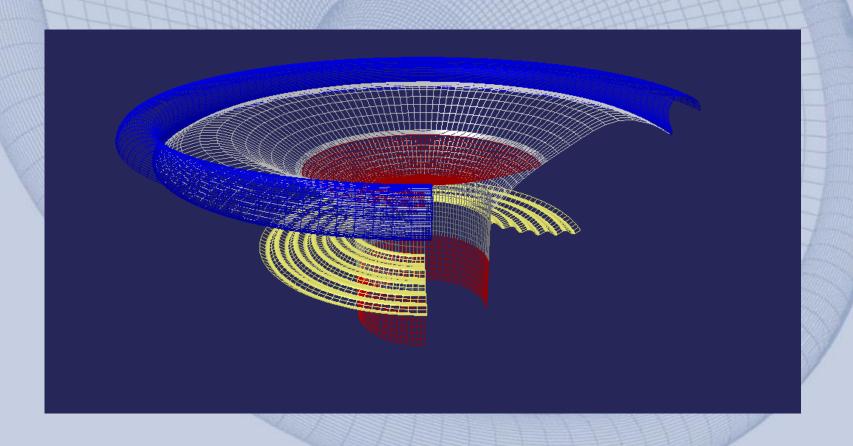
High order break-up of 6½" woofer @ 13623 Hz. The cone break-up has just reached the Voice Coil former.

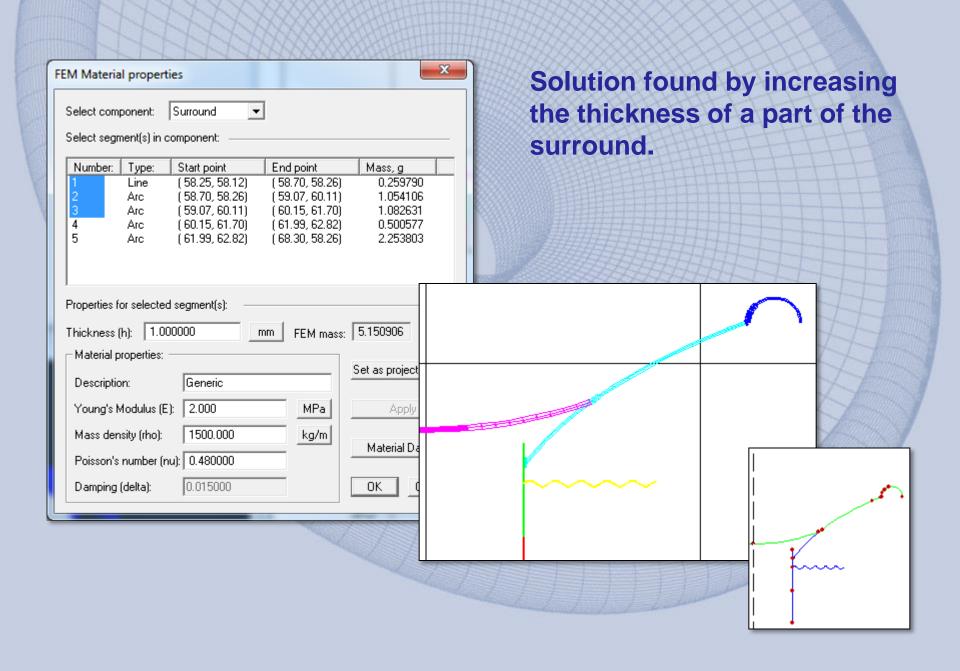


Example: 165mm/6.5" Woofer with a response problem 1000-1500 Hz

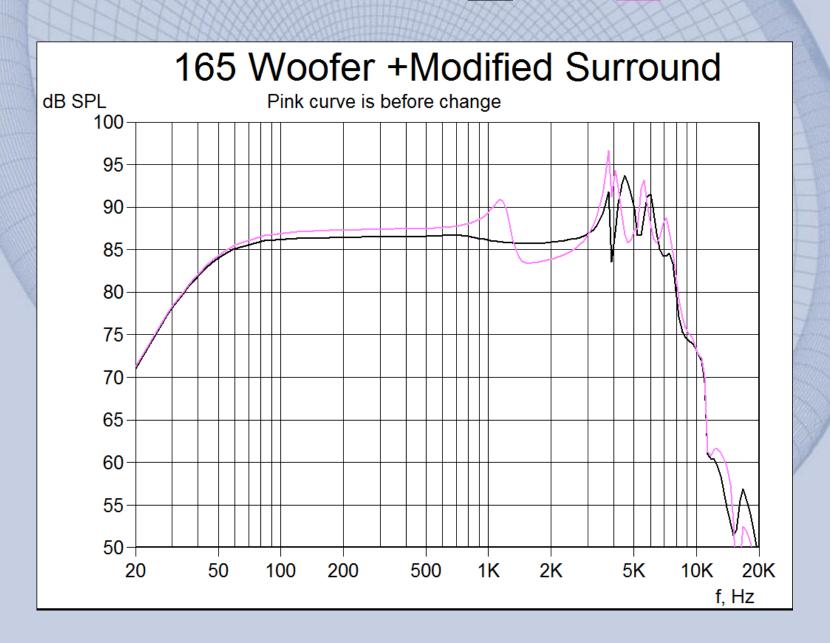


The FEM analysis reveals a strong edge resonance causing the problem around 1355 Hz

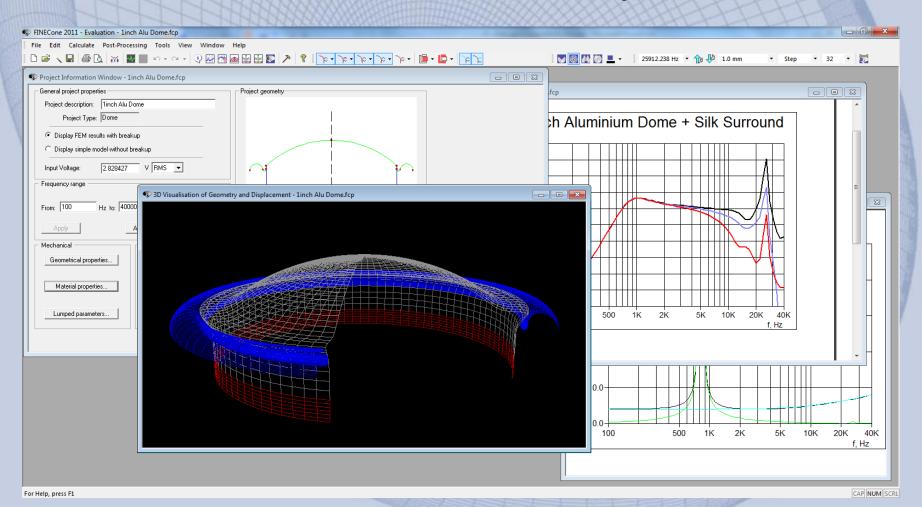




FEM simulated solution___/ before



FEM simulated 0__/30__/60__ deg. frequency responses of 1" Aluminium Dome Tweeter with break-up @ 25912 Hz.



Crossover Designs in Practice

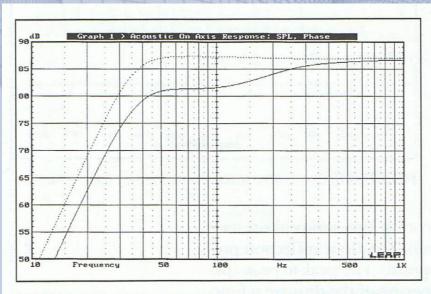


Crossover design is very simple in theory. In reality many problems makes it difficult and time consuming to design a good cross over circuit without the help of CAD.





Half space/baffle (2Pi) versus Anechoic (4Pi) Loudspeaker response



PRINT connent: Figure 4.12 Spreading loss for 6.5" MIM array

Enter connent or IAB for previous

MLSSA: Frequency Domain

FIGURE 4.11: Half-space versus anechoic response (dotted = half-space, solid = free-standing anechoic).

FIGURE 4.12: Spreading loss for 6.5" MTM array.

From "Testing Loudspeakers"

J. D'Apolito

Loudspeaker Measurements and Their Relationships with Listener **Preference**

Examples from Floyd Tool's article:

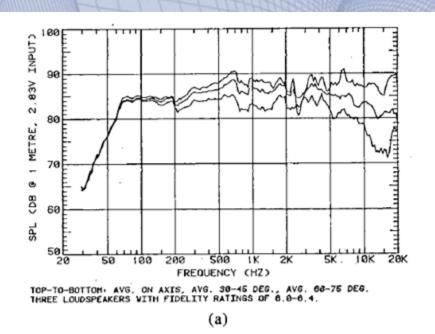
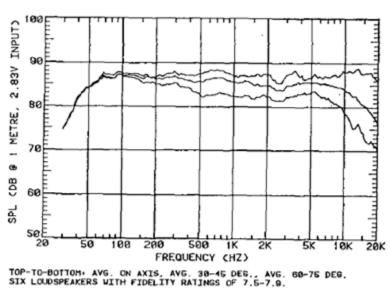


Fig. 7. Amplitude response measurements of loudspeakers with fidelity ratings (a) 6.0-6.4.

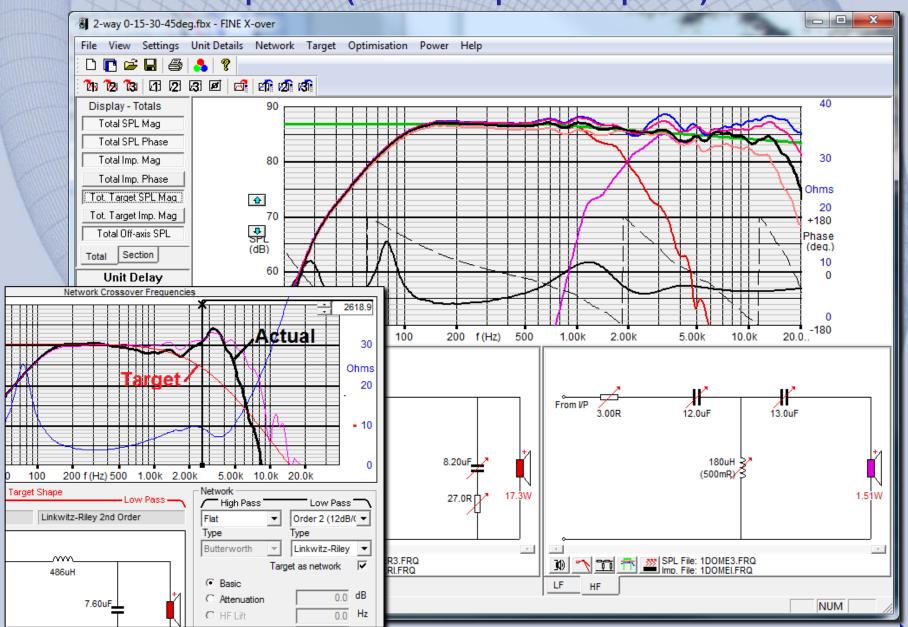


(d)

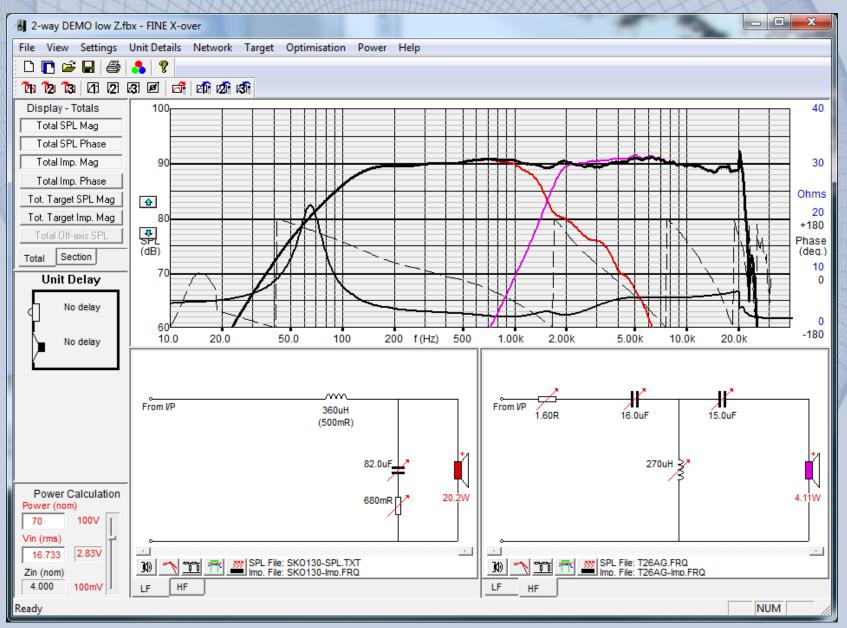
Fig. 7. Amplitude response measurements of loudspeakers with fidelity ratings (d) 7.5-7.9.

J. Audio Eng. Soc., Vol. 34, No. 5, 1986 May

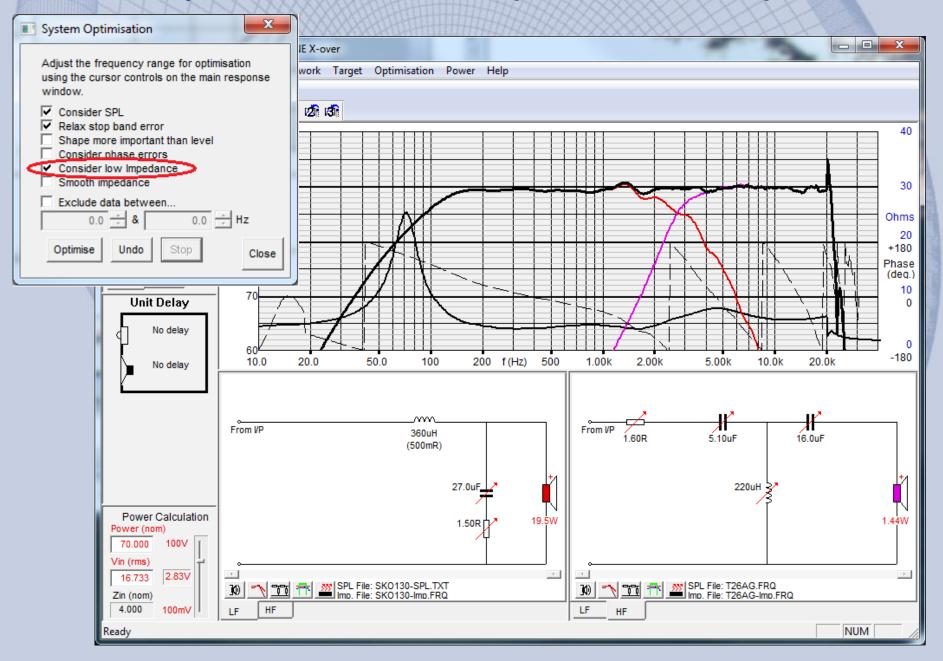
Example of 2-way cross over circuit with optimized off-axis responses (Controlled power response)



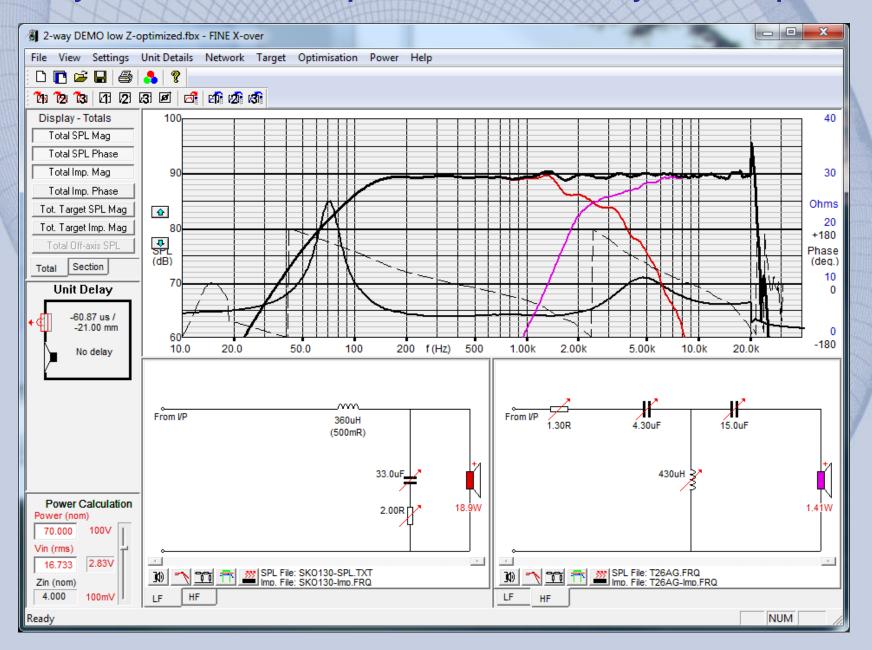
2-way cross over optimized for flat response. But Impedance is too low! (~2R)



2-way cross over circuit now optimized also for impedance Z



2-way cross over further optimized with unit delay for linear phase



Loudspeaker Measurement Examples



Some examples from using modern measuring methods

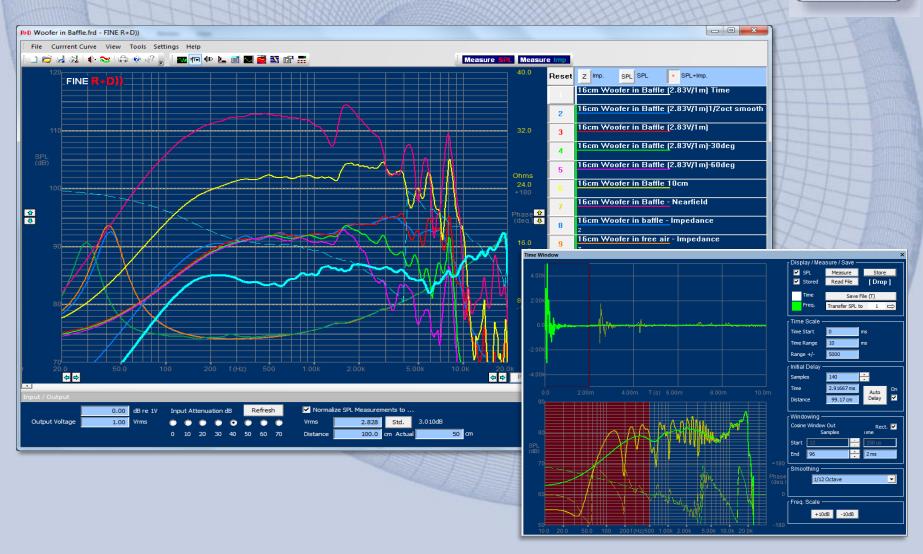


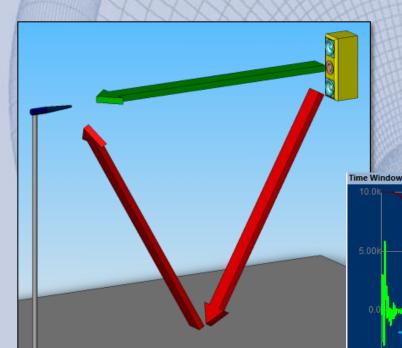




The swept sine / FFT technique makes it simple to get accurate frequency responses in normal rooms



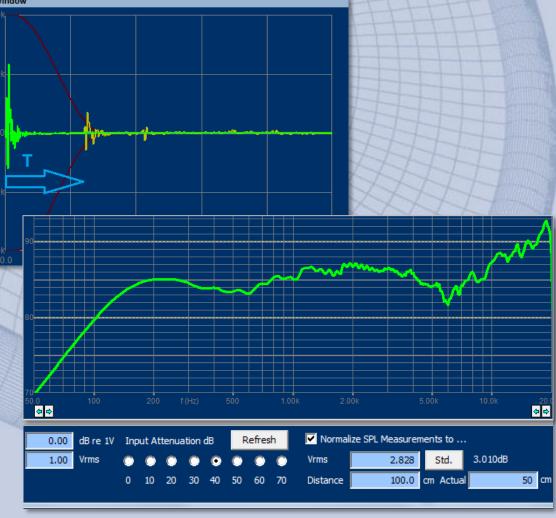




The time difference T between the direct__ and the reflected sound__ must be large in order to measure low frequencies: F min=1/T (Hz)

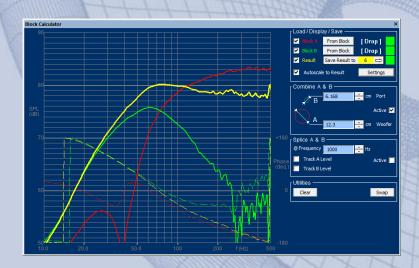
In a small room T can be increased by moving the microphone closer.

This curve measured at 1V/50cm was normalized to 2.83V/1m (=Industry standard)



LF Near field measurements:

Near field measurements will measure all low frequencies. However since these are really pressure responses it is necessary to compensate for the differences due to distance and piston size.



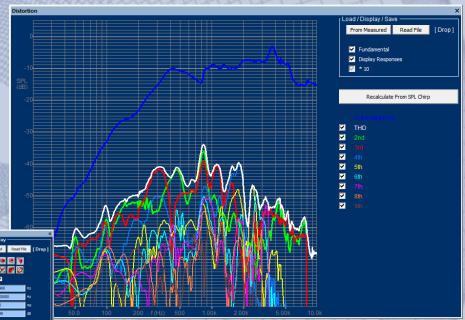
Bass reflex response , complex addition of woofer and port . Auto compensated for area differences

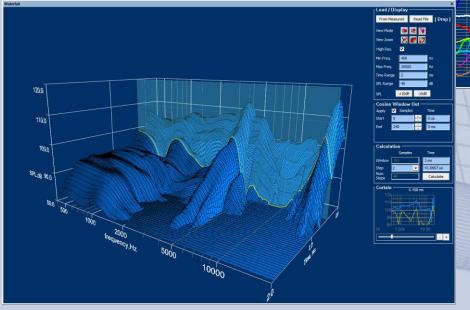
Total response , combined by far field and near field . Spliced @ 473 Hz



Harmonic distortion is useful. However the waterfall provides more information

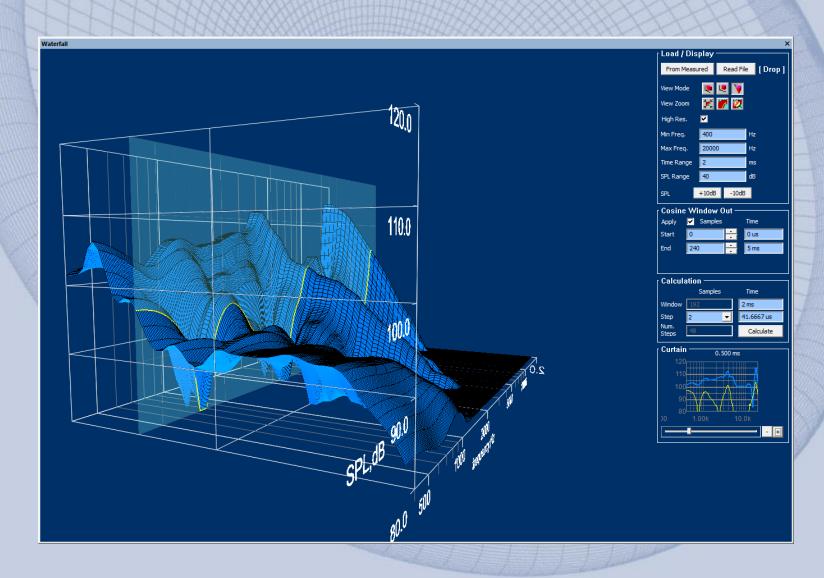
The 1-3rd harmonics are high @ 800 Hz.





The waterfall shows a strong reflection @ 800 Hz and a decaying resonance @ 4 kHz

The curtain shows the reflection @ 800 Hz in detail



Replacing old (DOS) Measuring systems

Old System: DOS



No Win 7 or 8 (+ 64bit)



No Amplifier
No Rub & Buzz



New System: (C++)



Challenges in Speaker QC Testing



A few notes about Speaker End of Line Quality Control in today's high speed production.

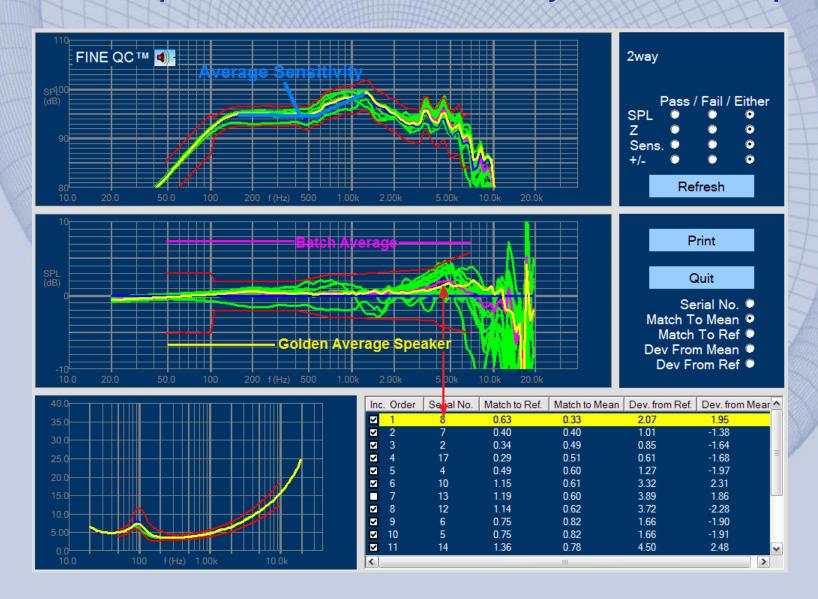




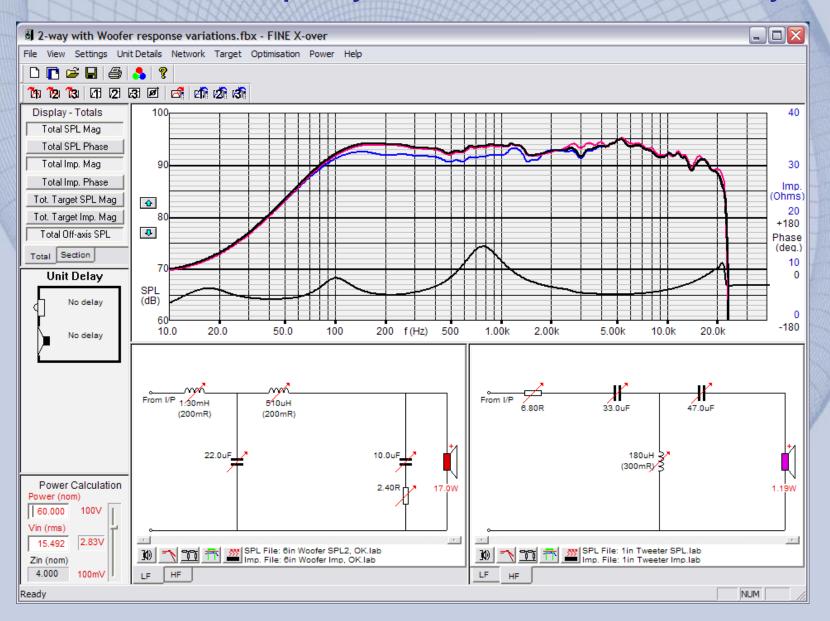
Which parameters should be tested to ensure good speakers and especially micro speakers, in production?



- Use statistics to find The Golden Average (REF)
- Decide response deviations as: Sensitivity and +/- x dB freq. band



Example: Two rejected woofer responses imported into FINE X-over. Red__ outside frequency tolerance. Blue__ low sensitivity



It is vital to find Rub & Buzz especially for micro speakers. This cannot be detected with conventional methods like THD, high harmonics or IM distortion.



How can Rub & Buzz be tested reliably?

Danish F. Leonhard derived in 1993 a new model for auditory perception based on mathematical and physical phenomena that correspond very much to how the human ear perceives sound.

A later detection method based on the Danish research principles uses a completely new protected algorithm to find the annoying sounds, which cannot be detected with conventional methods like THD, high harmonics or IM distortion. Finding fast low level impulses < -80 dB rel. signal is therefore possible.



How Accurate are Speaker Simulations?

Some results from AES 126th Loudspeaker FEA/BEM Workshop

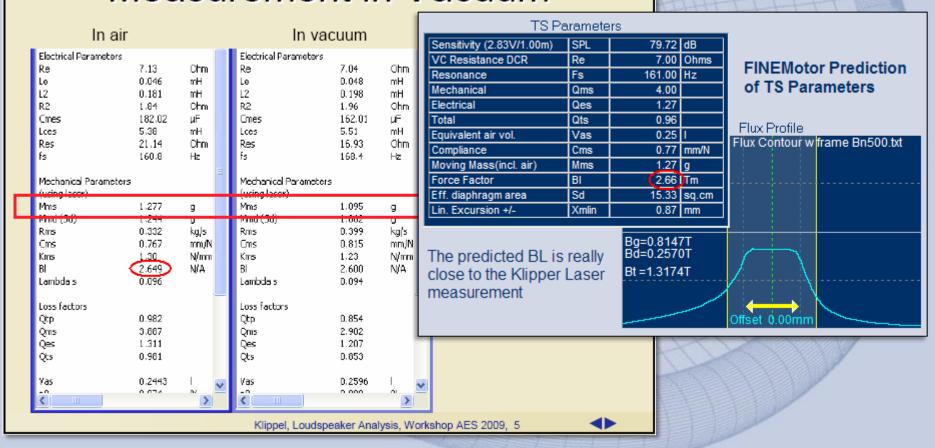




AES 126th Loudspeaker FEA/BEM Workshop

FINEMotor simulated TS parameters versus measured (Klippel)

Measurement in Vacuum

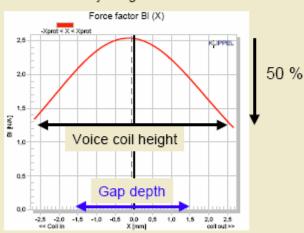


AES 126th Loudspeaker FEA/BEM Workshop

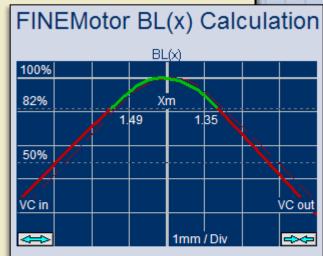
FINEMotor simulated BL(x) versus measured (Klippel)

Dominant Nonlinearities: Bl(x)

Measured by using the LSI module



XBI=1.3 mm @ Blmin=82 % (10 % intermodulation distortion in SPL, mainly 3rd-order)



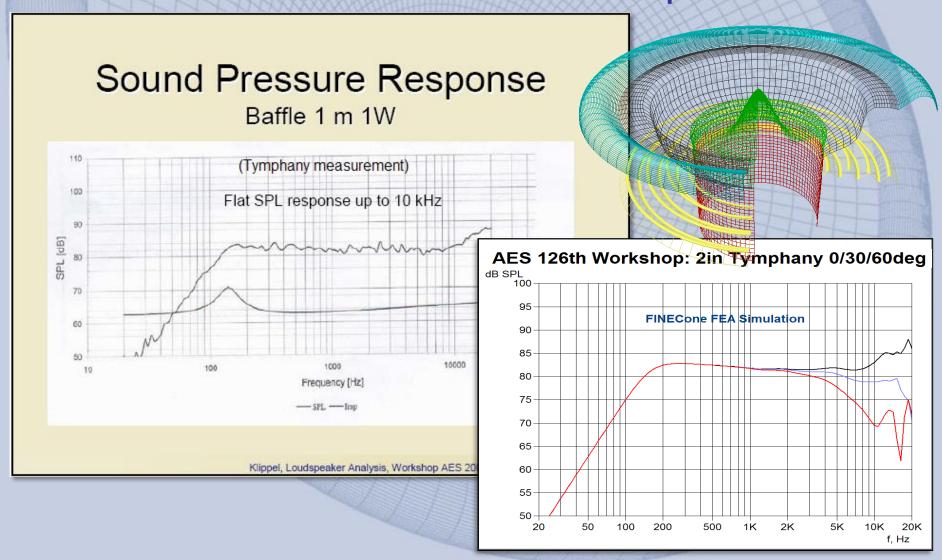
Predicted by FINEMotor:

XBL @ 82% BLmin =1.35mm. The shape and symmetry of the BL(x) curve is extremely close to that measured by Klippel

Klippel, Loudspeaker Analysis, Workshop AES 2009, 13

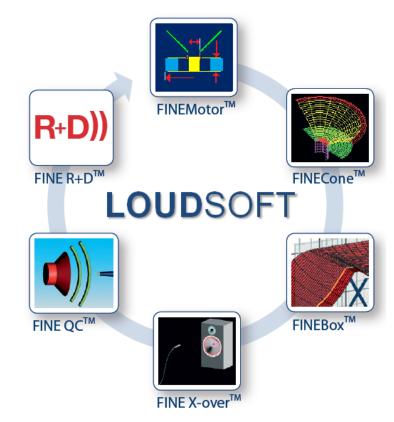


AES 126th Loudspeaker FEA/BEM Workshop FINECone FEA versus measured response





The FINE Circle



www.loudsoft.com



LOUDSOFT